

# *Tire testing at real driving conditions and at the test stand*

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#### Motivation

- Measurement Equipment
- Approach
- Test description
- Results
- Conclusions



#### Motivation



- The demand to higher efficiency concerns each component of future vehicles
- Tire resistance is one of the areas for efficiency improvements independent of vehicle drive concepts
- Understanding the behavior in real road conditions will become more important
- Standard testing methods (drum based) do not deliver road condition related information
- Tire resistance value is relative low
- Real road conditions measurement was suffering from:
  - Accurate measurement equipment for the forces
  - Ability to separate different influence sources
  - Low repeatability

#### Measurement Equipment on Road



- Vehicle measurement System (VMS)
  - -Wheel Force Sensor(WFS)
  - Wheel Position Sensor (WPS)
  - Other sensors such as GPS
  - Vehicle ECU Information



## Rig Measurement Equipment



- Flat belt tire testing rig (steel belt)
  - Best simulation of the road
- Test is performed with the same sensor used for the vehicle testing

Rig Specification	
Velocity	0~200km/h
Slip Angle	±20deg(0~3Hz)
Camber Angle	±15deg(0~1Hz)
Up & Down	0~50mm(0~25Hz
Load	Fx: ±10 kN Fy: ± 10 kN Fz: 12 kN
Flatness of the steel belt (under load condition)	Less than 10 µm
Bearing under the belt	Air bearing



#### Wheel Force Sensor (WFS)



6 component in wheel force sensor main properties

- 3 axis of force and 3 axis of moment
- 0.1% Resolution
  - -6N or 1.8Nm
- Capacity:
  - -Fx = 24KN, Fy = 15KN, Fz = 24KN
  - Mx= 4.5 KNm, My =4 KNm, Mz = 4.5KNm
- Data acquisition up to 1kHz
- Lightweight 3.2 Kg



## Unique Force Detection Method

- Model Based Sensor concept
- Shared force detection method
  - Eight bridges are applied to the spring element
    - No direct detection of each component
  - Components are re-composed by model based calculation using real time calculation DSP platform
  - Digital conversion of all signals and electronically re-composing overcomes disadvantages of analogue approach
  - Cross talk error can be canceled out



#### Minimized Temperature effects



- Vehicle measurement is a challenge for the temperature influence
  - Temperature gradient e.g. break side outside
  - -Quick change of temperature depending on driving maneuver
- Robust design against Temperature effects
  - Share Force method allows to place the strain gauges very close to each other
  - Total gradient on each gauges is very small
  - Small temperature effect on the measurement
  - At the same time robustness against dynamic temperature changes



#### Mechanical and Electrical sensitivity



- Stiff sensor design for high accuracy
- Sensor sensitivity:
  - Mechanical sensitivity x electrical sensitivity
- Stiff Spring element design
  - Increase of robustness
  - Increase of eigenfrequency
  - Reduction of mechanical sensitivity
- Increase electrical sensitivity
  - High precision A/D converting of nV order A/D conversion
  - Low noise design from less analog circuit
  - Optimized temperature compensation from gauge layout
- The combination of all technology results in a high accurate sensor with 1/4000 resolution

#### Wheel Force Sensor Configuration





## Tire Loss Theory



- Tire loss can be calculated from measured parameters on the wheel
- Measurement parameters
- Tire rolling inertia
   Jt in kg·m<sup>2</sup>
- Tire effective radius
   r<sub>t</sub> in m
- Wheel torque My in Nm
- Tire longitudinal force Fx in N
- Tire Angular speed  $\omega$  in rad/s
- Tire Angular acceleration ώ rad/s<sup>2</sup>
- Calculated parameter
  - Tire loss (rolling resistance) Rx in N

 $R x = \frac{My + Jt * \dot{\omega}}{r_t} - Fx$ 

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Driven Wheel

#### Parameter determination: Wheel inertia



- Tire rolling inertia is premeasured using free load rotating wheel in acceleration and deceleration condition
  - Measurement items
    - Tire angular speed ω [rad/s]
    - Angular acceleration ώ [rad/s<sup>2</sup>]
    - Wheel torque My<sub>free</sub> [Nm]
  - Rolling inertia formula:





- 50 50

0

AngleAccele [rad/s<sup>2</sup>]

50

100

## Testing procedure on the test track

- Target: Determine "Tire Loss" from real driving condition
- Test car: BMW Mini Cooper S
- Test Track:
- Total length: 1,792m
- East straight line: 550m
- West straight line: 554m
- Driving Maneuver:
  - Acceleration at west straight line
  - Cost down at East straight line
  - Test laps: 10 laps
- 100Hz data acquisition



#### Test Track Measurement Results





#### Angular acceleration determination

- Tire angular speed is measured from sensor angle encoder.
- Tire angular acceleration is calculated from angular speed signal by time derivative
- Measurement item:
  - Tire angular speed
     ω [rad/s]

 $\dot{\omega} = \frac{\mathrm{d}\omega}{\mathrm{d}t} \left[ rad / s^2 \right]$ 

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Tire angular acceleration







#### Tire radius determination



- Tire mean radius is calculated from vehicle velocity and tire angular speed.
- Vehicle velocity is measured from optical Doppler sensor
- Instant tire mean radius is measured.
- Measurement items
  - Vehicle velocity against road Vph [m/s]
  - Tire angular speed
     ω [rad/s]
- Tire radius formula

   (Not considering tire slip)
   rt = <u>Vph</u> [m]



 $\omega$ 

## Measurement parameter: Wheel torque and longitudinal force



#### Tire Loss determination (Rolling Resistance)



#### Rear Left Wheel results

![](_page_18_Picture_1.jpeg)

- Average Rx: Rx = -76.1N (Acceleration), Rx = -72.8N (Cost down)
- 10 laps data variation 3σ : 2.8N (Acceleration), 3.6N (Cost down)
- Rx for Acceleration and Rx for Cost down data are very close to each other: 3.3N

![](_page_18_Figure_5.jpeg)

#### Rear Right Wheel results

![](_page_19_Picture_1.jpeg)

- Average Rx: Rx = -87.6N (Acceleration). Rx = -82.6N (Cost down)
- 10 laps data variation 3σ: 2.5N (Acceleration)., 6.6N (Cost down)
- Rx for Acceleration and Rx for Cost down data are very close to each other: 5.0N

![](_page_19_Figure_5.jpeg)

#### Measurement result : Test rig

![](_page_20_Picture_1.jpeg)

Wheel Load & Tire Rotational Speed 6000 Test condition Load Speed ||150 4000 Speed[rad/s] • Slip angle :0 [deg] Force[N] 2000 00 Camber angle: 0 [deg] • Wheel driven by steel belt - 2000 • Vertical load Fz: 1kN, 2kN, 5kN 100 50 Time[s] • Static velocity: 5km/h, 10km/h, **Rolling Resistance** 20km/h, 60km/h, 80km/h, 120km/h Resistance[N] <del>∃ B</del> Fz1kN - 20 ⊖ ⊖ Fz2kN Rolling resistance is directly measured 🔺 🔺 Fz5kN - 40 from Fx - 60 Rolling resistance is proportional to the vertical load and is not a function of - 80 10 100  $1 \times 10^{3}$ Velocity[km/h] velocity **Rolling Resistance**  Rolling resistance at 2.7kN is 42N <u>→</u> 005km/h <del>) ⊖</del> 010km/h Resistance[N] 🛆 020km/h 040km/h 080km/h ▲▲ 120km/h - 80 1000 2000 3000 4000 5000 Fz[N] 7. Intelligent Tire Technology

#### Comparison: Real road vs Test rig

![](_page_21_Picture_1.jpeg)

- Real road rolling resistance :
- Rx(Left) = 74 N
- Rx(Right) = 82 N

Test rig:

• Rx = 42 N

Reasons for the difference:

- Tire alignment
- Road surface condition
- Environment conditions
  - Wind force to tire
  - Temperature
- Measurement errors
  - Tire effective radius measurement

![](_page_21_Picture_15.jpeg)

![](_page_21_Picture_16.jpeg)

#### Conclusion

![](_page_22_Picture_1.jpeg)

- A&D Sensor delivers high quality data
- It was possible to measure the tire loss (rolling resistance) during real driving condition
- Great match on the measurement though 10 laps of data
- Rolling resistance measurement result is depending on driving conditions
  - There is a different between acceleration and cost down conditions
- Useful measurement for analyzing energy loss value at real driving condition
- There are differences between road and test rig results

![](_page_23_Picture_0.jpeg)

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## Thank you for your attention!

Measurement System

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**EWPS**